

Dans le cadre de la
Formation « Hydraulique Fondamentale »
Unité de Formation pour la Performance Industrielle



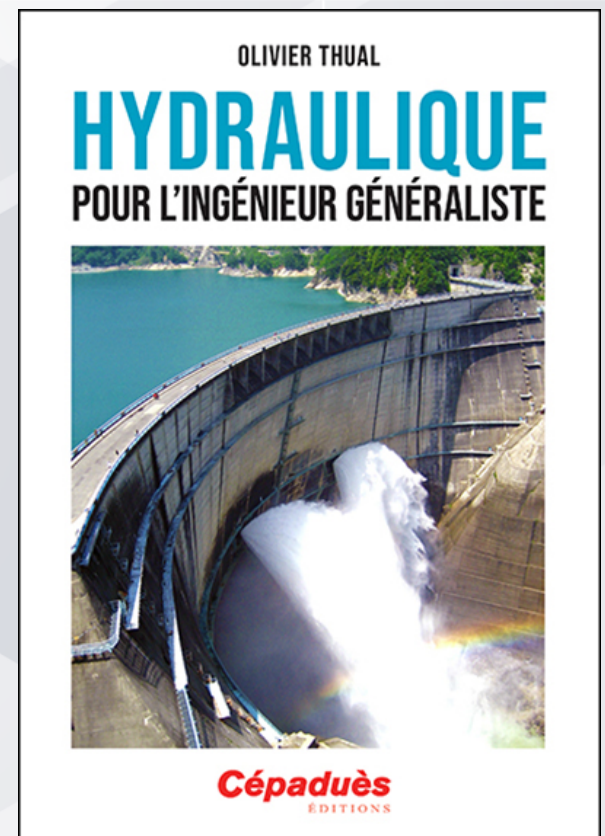
en collaboration avec l'INP-ENSEEIH

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Chapitre 3

Pertes de charge



Contrainte pariétale

$$\frac{dH}{ds} = -S_f = -\frac{\Phi_f}{\rho g}$$

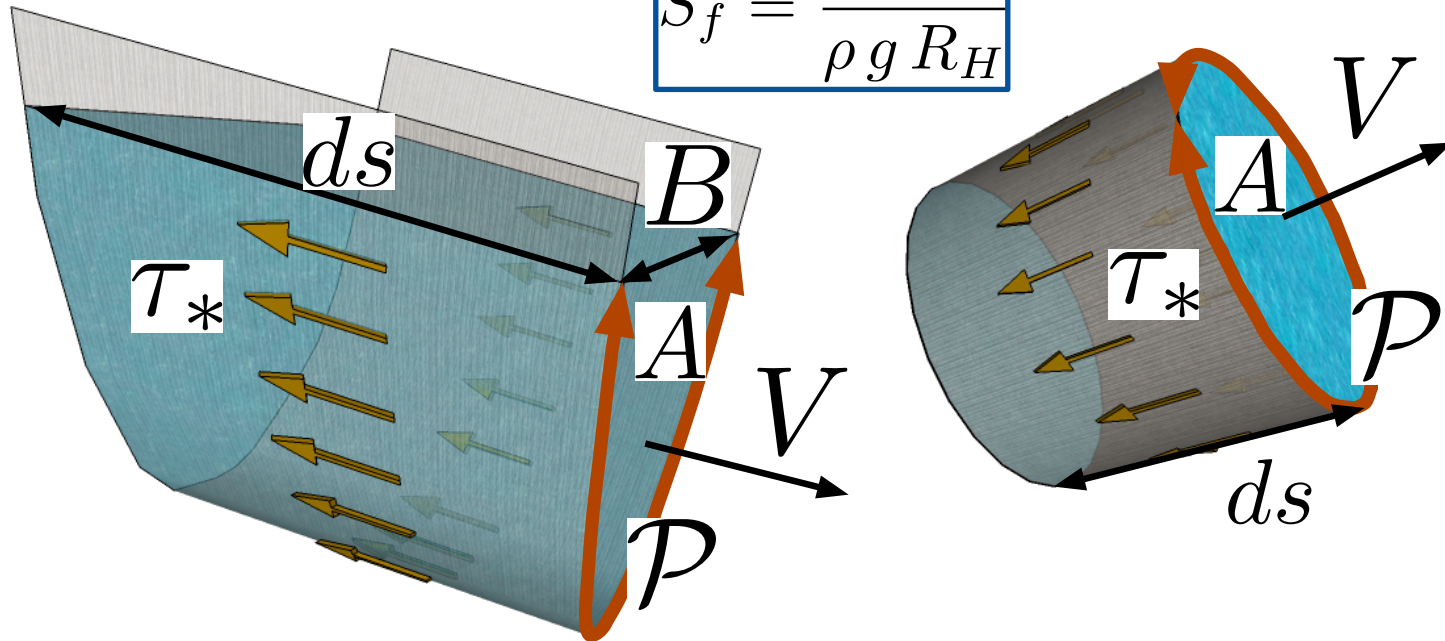
$$\tau_* \mathcal{P} ds = \Phi_f A ds \iff \tau_* = \Phi_f R_H$$

$$\tau_* = \rho g R_H S_f$$

Rayon hydraulique :

$$R_H = \frac{A}{\mathcal{P}}$$

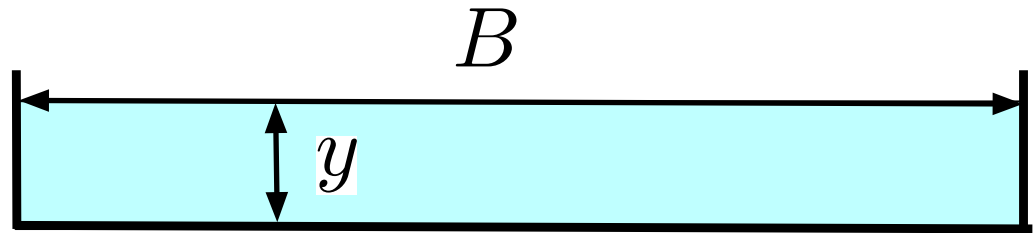
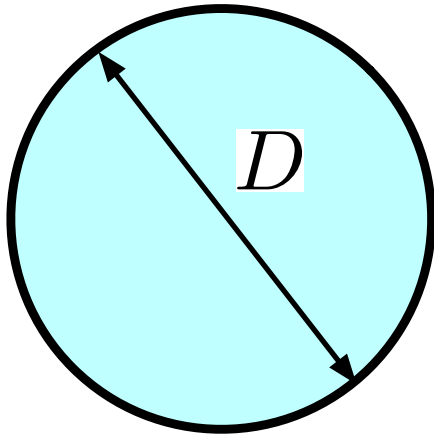
$$S_f = \frac{\tau_*}{\rho g R_H}$$



Rayon et diamètre hydrauliques

$$R_H = \frac{A}{P}$$

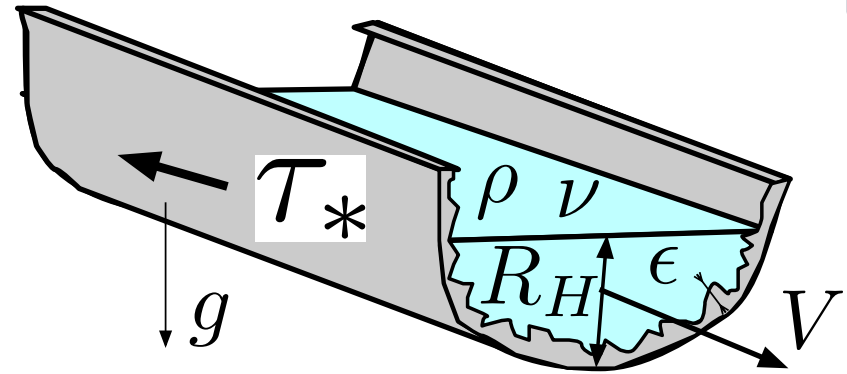
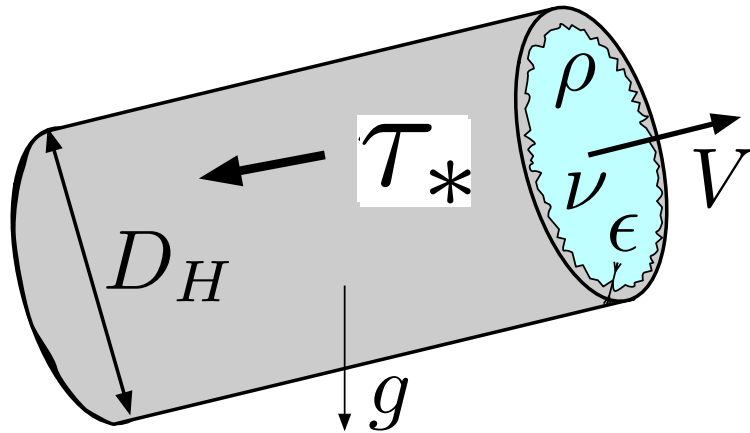
$$D_H = 4 R_H$$



$$D_H = D$$

$$R_H = y \quad \text{si} \quad y \ll B$$

Coefficient de frottement



Cinq paramètres : $(\epsilon, D_H, V, \nu, g)$

Deux unités : (m, s)

Rugosité relative

Nombre de Reynolds

Nombre de Froude

Trois nombres sans dimension :

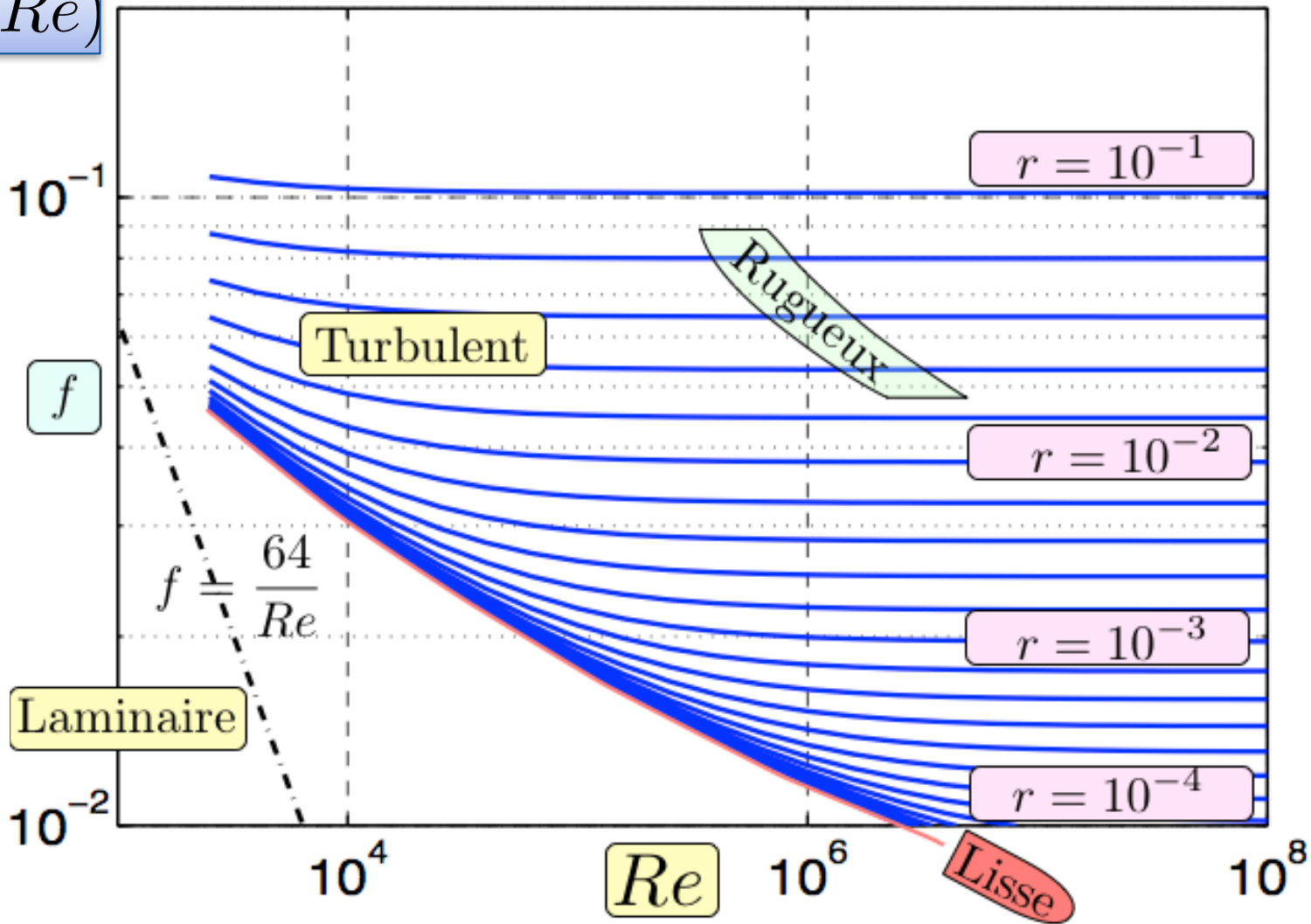
$$\left(r = \frac{\epsilon}{D_H}, \quad Re = \frac{V D_H}{\nu}, \quad Fr = \frac{2V}{\sqrt{g D_H}} \right)$$

La contrainte pariétale ne dépend pas de la gravité :

$$\tau_* = \frac{1}{8} f(r, Re) \rho V^2$$

Diagramme de Moody

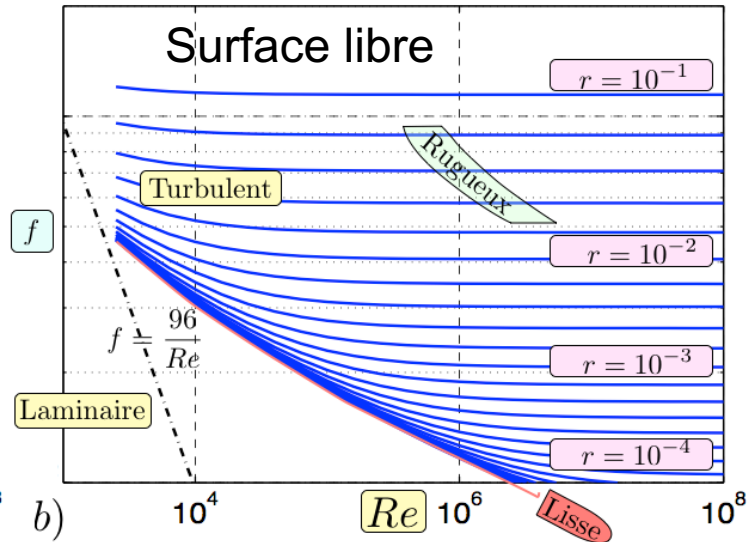
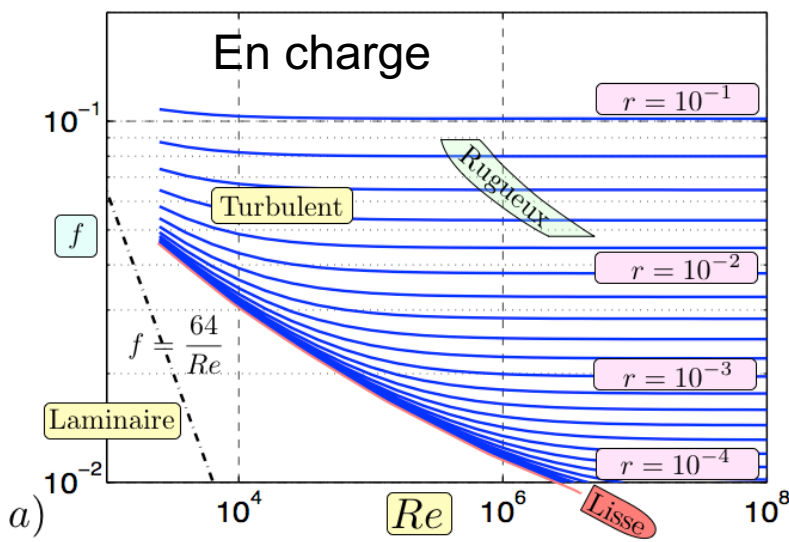
$$f(r, Re)$$



Relation de Darcy-Weisbach

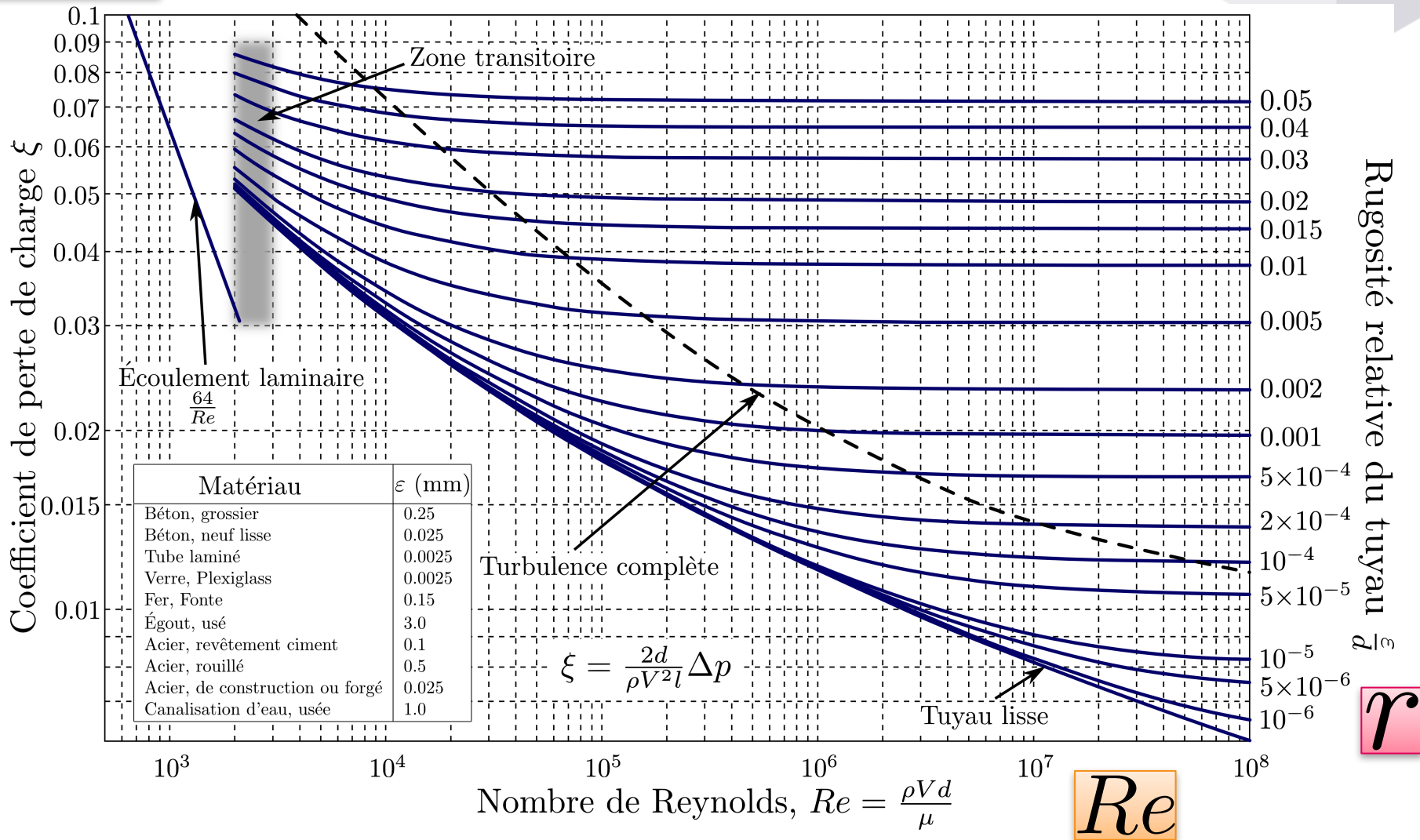
$$\frac{dH}{ds} = -S_f, \quad S_f = \frac{\tau_*}{\rho g R_H}, \quad D_H = 4 R_H \quad \text{et} \quad \tau_* = \frac{1}{8} f(r, Re) \rho V^2$$

$$-\frac{dH}{ds} = S_f = f(r, Re) \frac{V^2}{2 g D_H} \quad \text{avec} \quad r = \frac{\epsilon}{D_H} \quad \text{et} \quad Re = \frac{V D_H}{\nu}$$

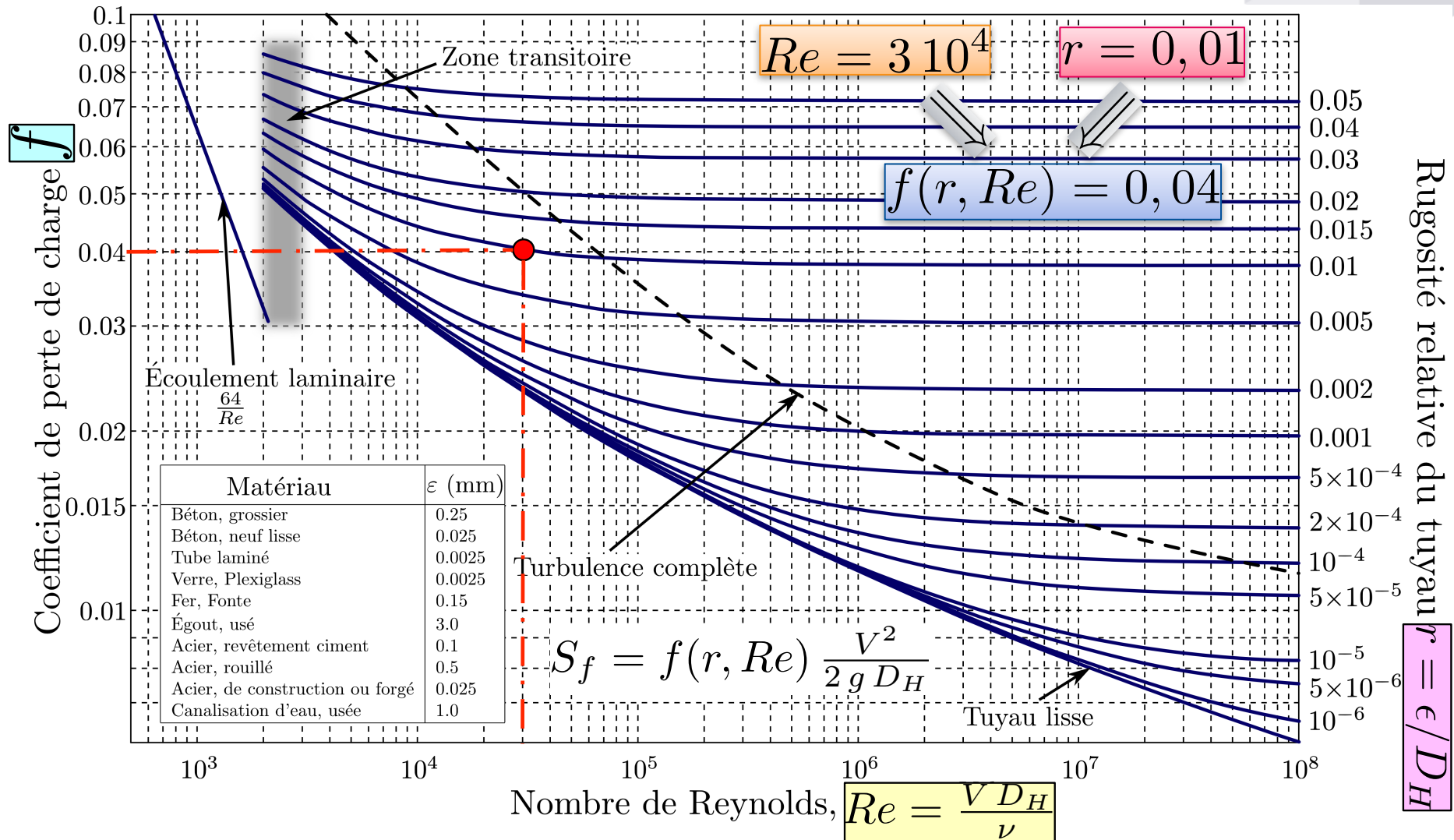


Conduites circulaires

$$f(r, Re)$$



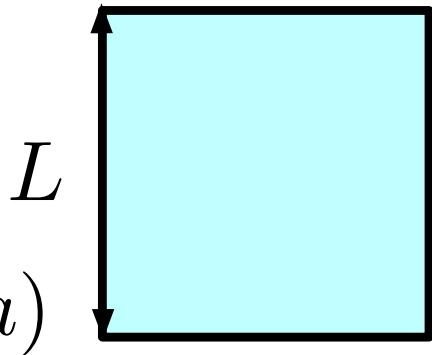
Lecture du diagramme



Rayon et diamètre hydrauliques

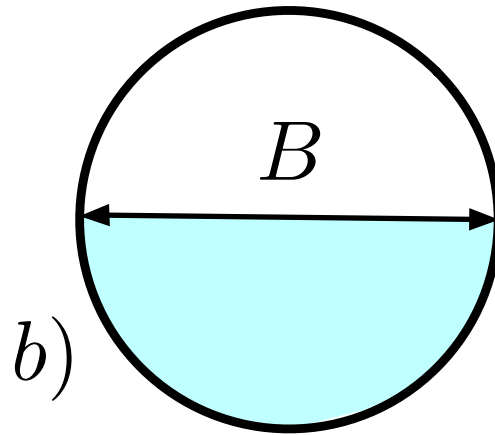
$$R_H = \frac{A}{\mathcal{P}}$$

$$D_H = 4 R_H$$



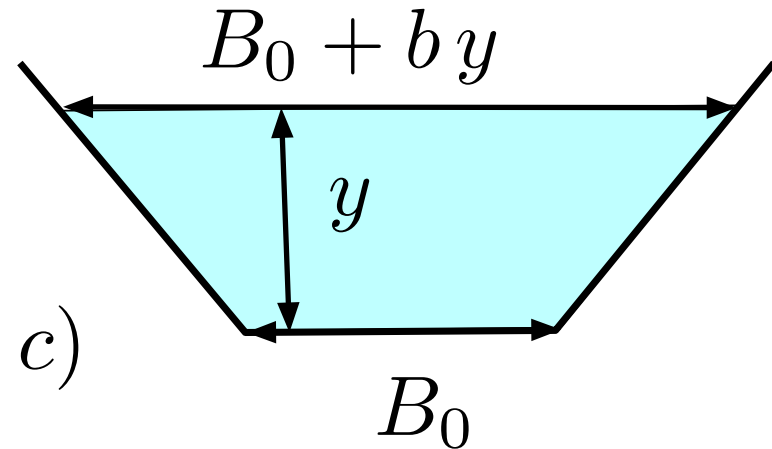
$$D_H = L$$

$$R_H = L/4$$



$$D_H = B$$

$$R_H = B/4$$



$$R_H = y \frac{1 + b \frac{\xi}{2}}{1 + 2 \xi \sqrt{1 + \frac{b^2}{4}}}$$

avec $\xi = \frac{y}{B_0}$